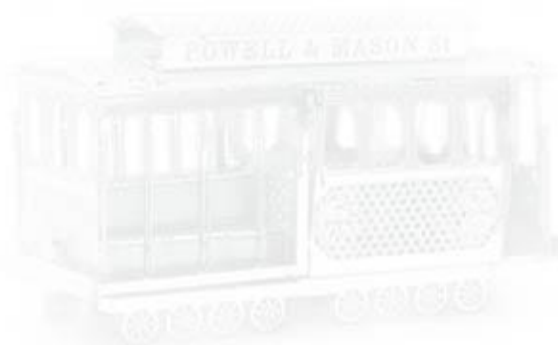


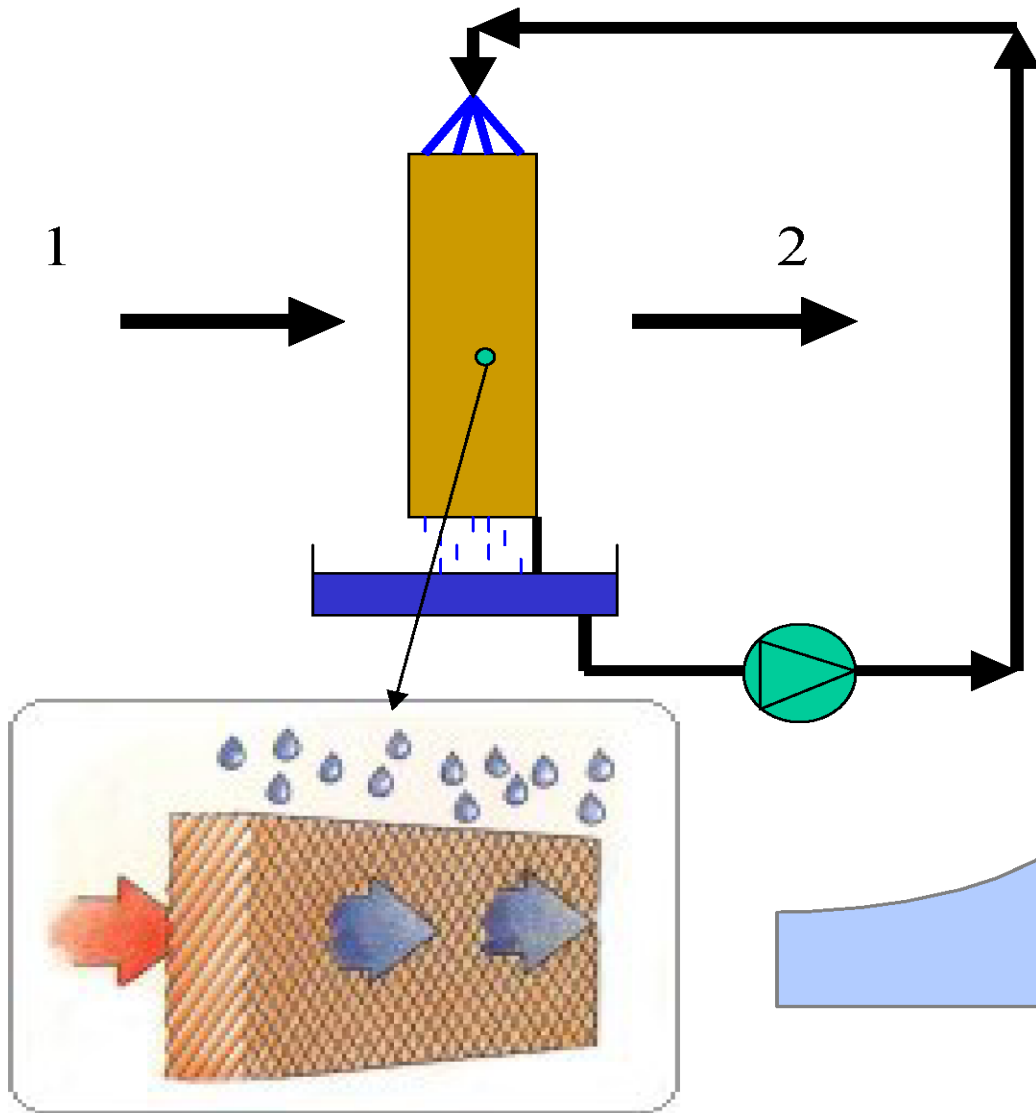


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EPX Heat Exchanger Technology for Dry Evaporative Cooling

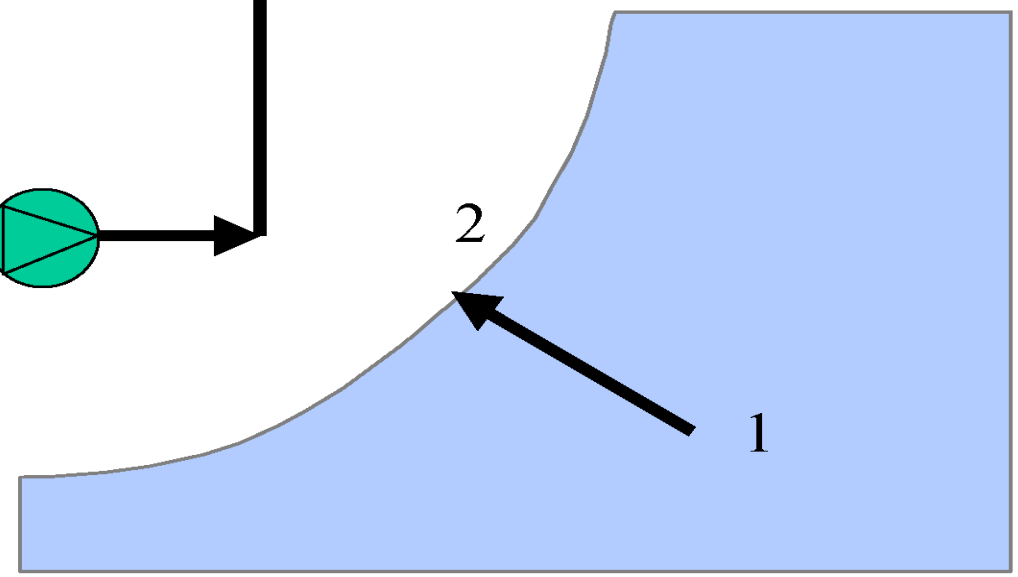
Vijayanand Periannan

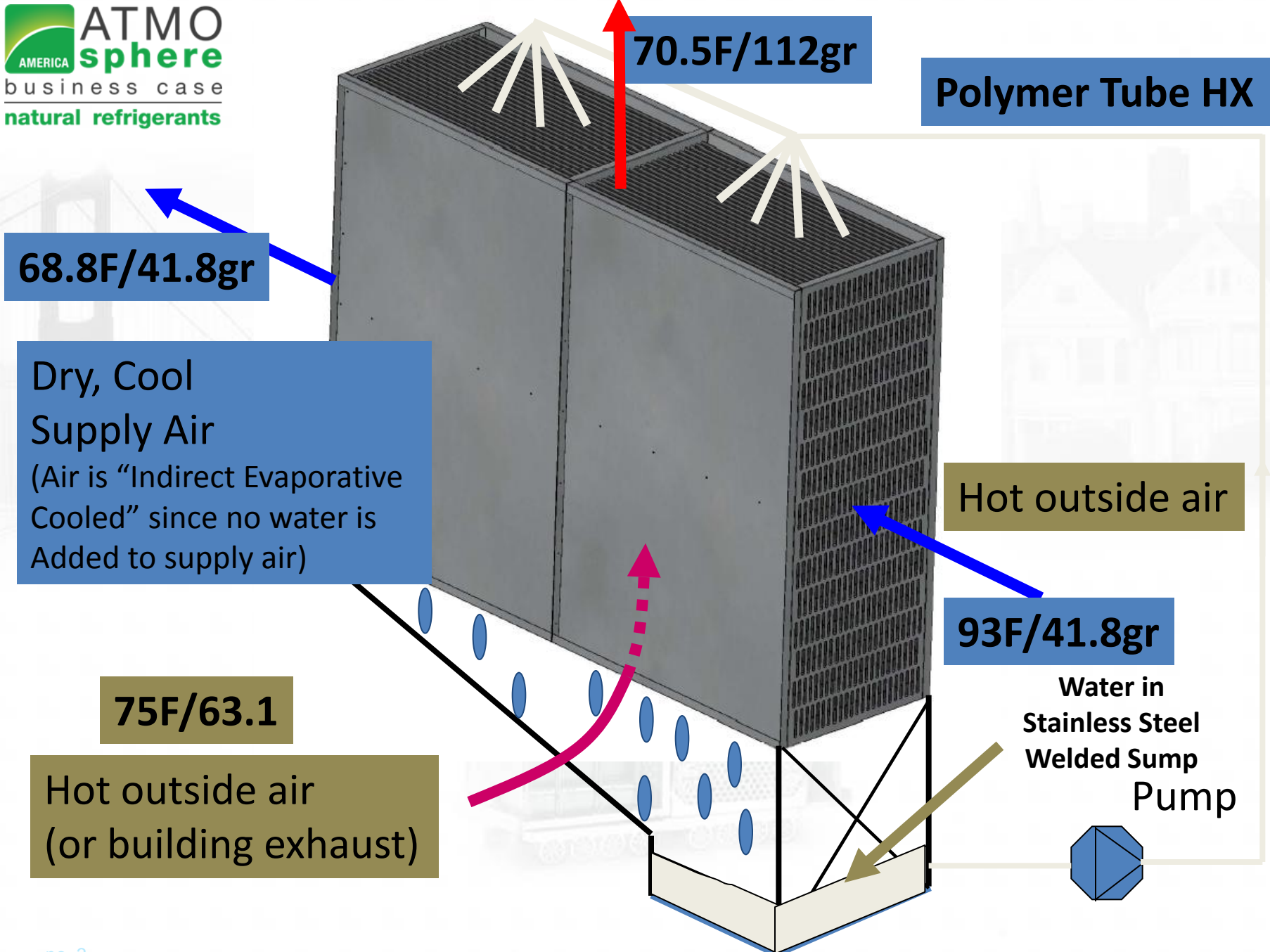




DIRECT EVAPORATIVE
 COOLING ... A CONSTANT
 WET BULB PROCESS

HUMIDIFIES AND COOLS





70.5F/112gr

Polymer Tube HX

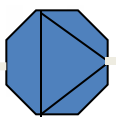
68.8F/41.8gr

Dry, Cool
Supply Air
(Air is "Indirect Evaporative
Cooled" since no water is
Added to supply air)

Hot outside air

93F/41.8gr

Water in
Stainless Steel
Welded Sump
Pump



75F/63.1

Hot outside air
(or building exhaust)

Wet Bulb Depression Efficiency(WBDE)

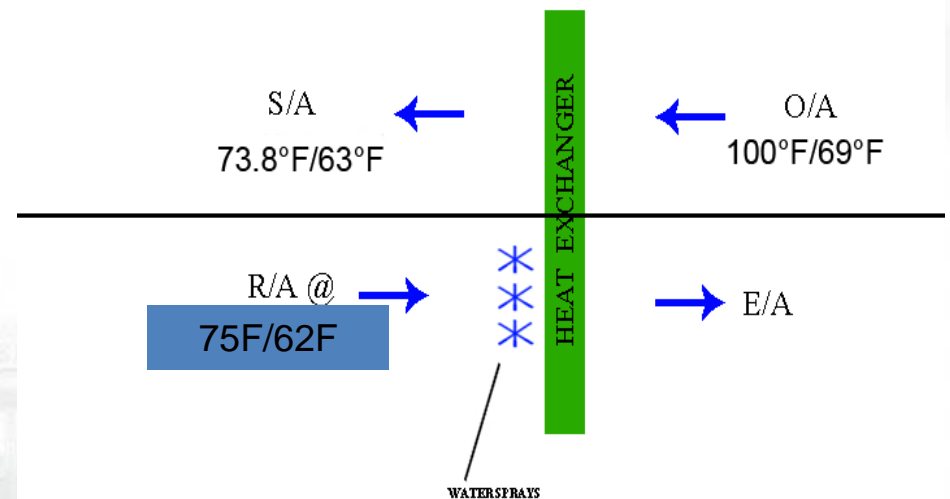
$$\text{Effectiveness} = 100\% \times \frac{(\text{EDBT}-\text{LDBT})}{(\text{EDBT}-\text{WBT})}$$

EDBT = Entering dry bulb temperature of primary air

LDBT = Leaving dry bulb temperature of primary air

WBT = Entering wet bulb temperature of secondary air

IEC WITH DIRECT SPRAYS IN THE R/A STREAM



- More cooling than dry air to air HX

Sample EER Calculation

| | |
|-------------------------------|---------------|
| CFM | 10000 |
| IEC EAT | 100 |
| R/A WB | 62.5 |
| WBDE | 0.7 |
| HP LAT (deg F) | 73.75 |
| dt | 26.25 |
| IEC cooling (btu/hr) = | 283500 |

| | |
|--|---------------|
| Net Total Cooling Capacity (btu/hr) | 283500 |
|--|---------------|

| | |
|-------------|--------|
| IEC Pump HP | 1 |
| IEC Pump KW | 0.7457 |

| | |
|--------------------------------------|------|
| Supply air pressure drop from IEC | 0.55 |
| Supply fan BHP contribution from IEC | 1.33 |
| Supply fan motor eff | 0.9 |
| Supply fan KW contribution | 1.10 |

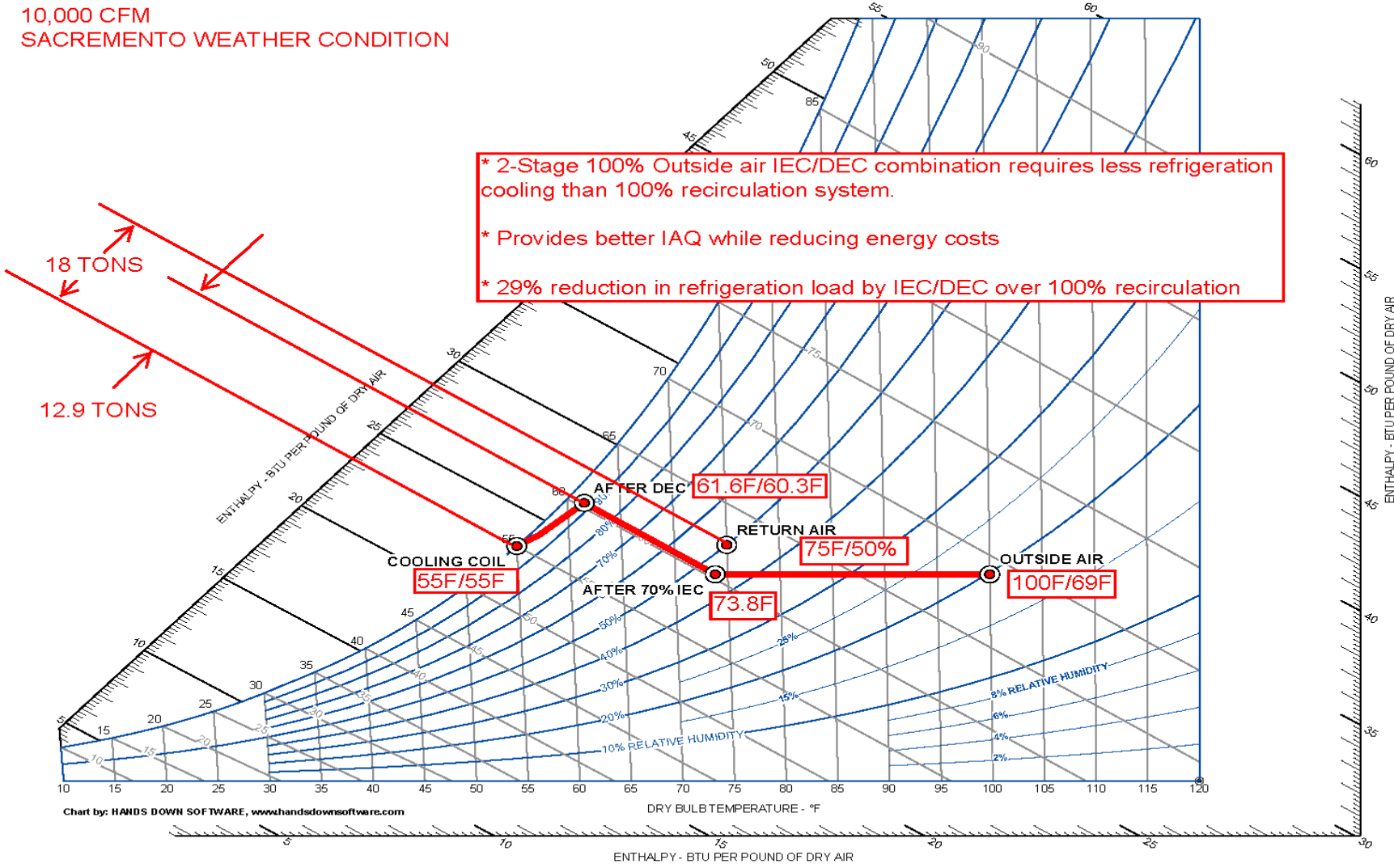
| | |
|---|------|
| Exhaust air pressure drop from IEC (wet side) | 0.45 |
| Exhaust fan BHP contribution from IEC | 1.09 |
| Exhaust fan motor eff | 0.9 |
| Exhaust fan KW contribution from IEC | 0.90 |

| | |
|--|-------------|
| Total Electric input to achieve cooling effect (KW) | 2.75 |
|--|-------------|

| | |
|------------|--------------|
| EER | 103.0 |
|------------|--------------|

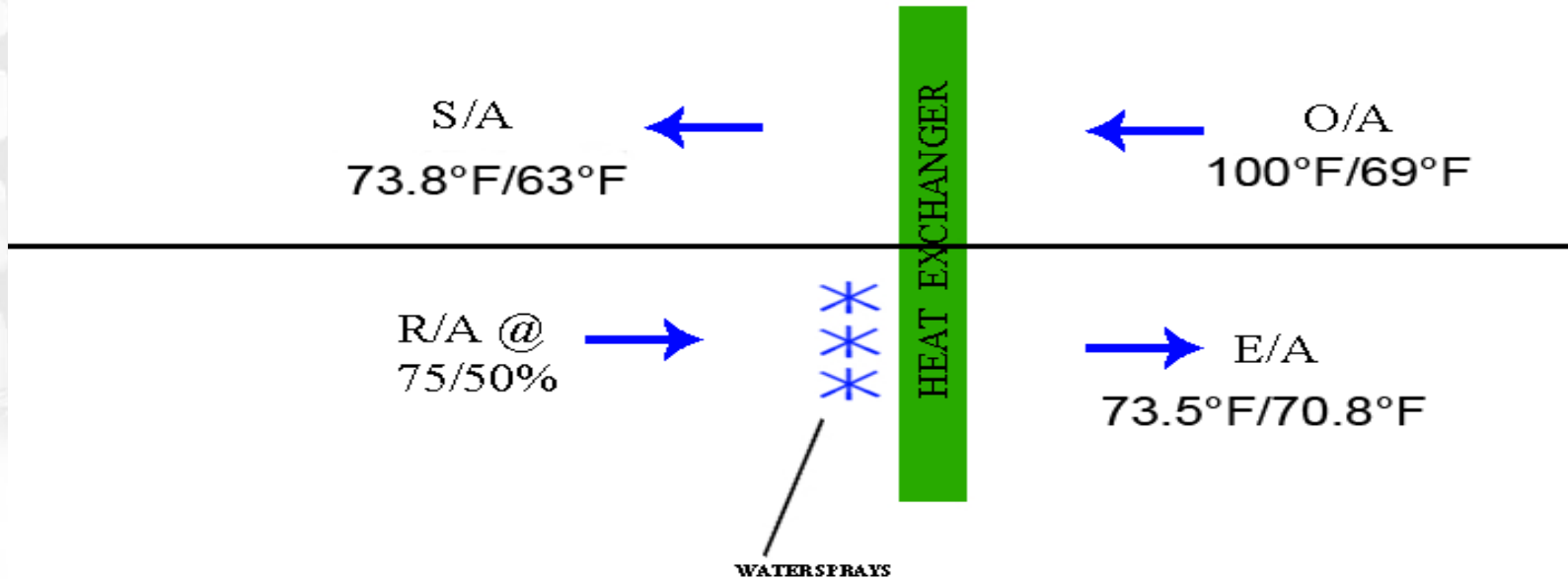
Combining IEC and DEC- IDEC Systems

100% O/A IEC/DEC VS 100% RECIRC SYSTEM
 10,000 CFM
 SACRAMENTO WEATHER CONDITION

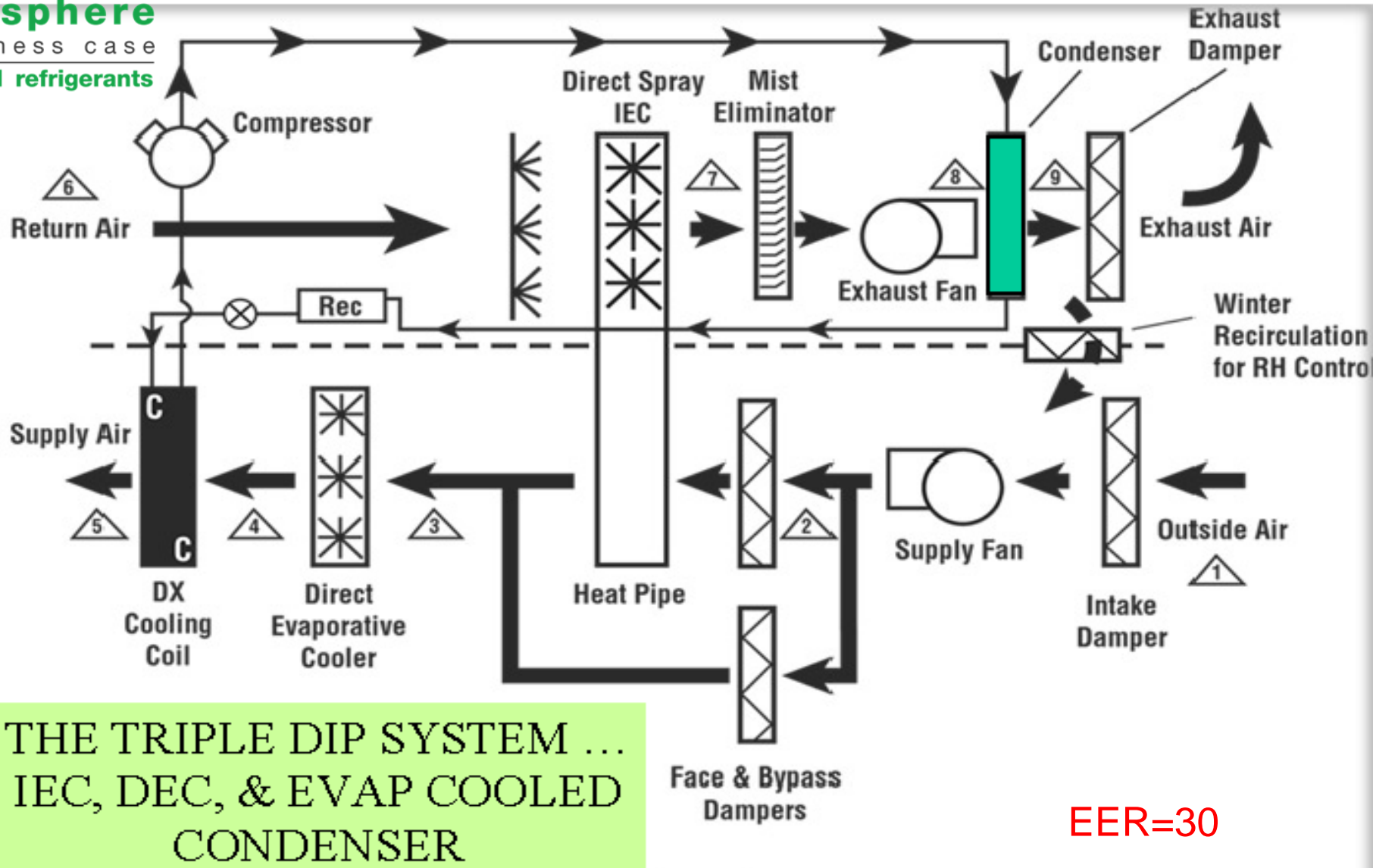


- * 2-Stage 100% Outside air IEC/DEC combination requires less refrigeration cooling than 100% recirculation system.
- * Provides better IAQ while reducing energy costs
- * 29% reduction in refrigeration load by IEC/DEC over 100% recirculation

Triple Dip System



- Humid but cool building exhaust air may be used for rejection of “Heat of Compression” for onboard DX refrigeration system.



THE TRIPLE DIP SYSTEM ...
 IEC, DEC, & EVAP COOLED
 CONDENSER

EER=30

Figure 1

ANNUAL ENERGY CONSUMPTION ANALYSIS

100% OUTDOOR AIR IDEC VAV SYSTEM

VS.

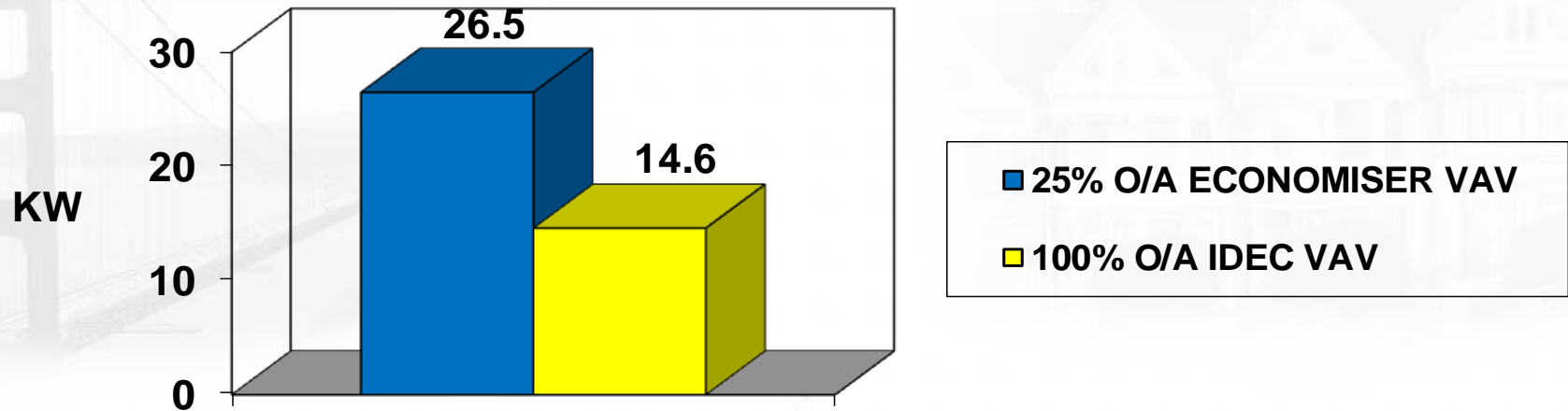
25% OUTDOOR AIR ECONOMISER VAV SYSTEM

PARAMETERS USED

- 10,000 CFM supply.
- Sacramento TMY3 data
- Supply air condition @ 55°F .
- Summer room temperature is 75°F and 50% RH.
- 7am-8pm/365days duty cycle.
- Winter room temperature is 70°F.
- IEC WBDE is 70%.
- DEC WBDE is 90%.
- 0.8 kW/ton chiller including auxiliary energy
- Winter heat recovery effectiveness is 50%.
- VAV minimum turndown is 25% at ambient conditions 55F and below.

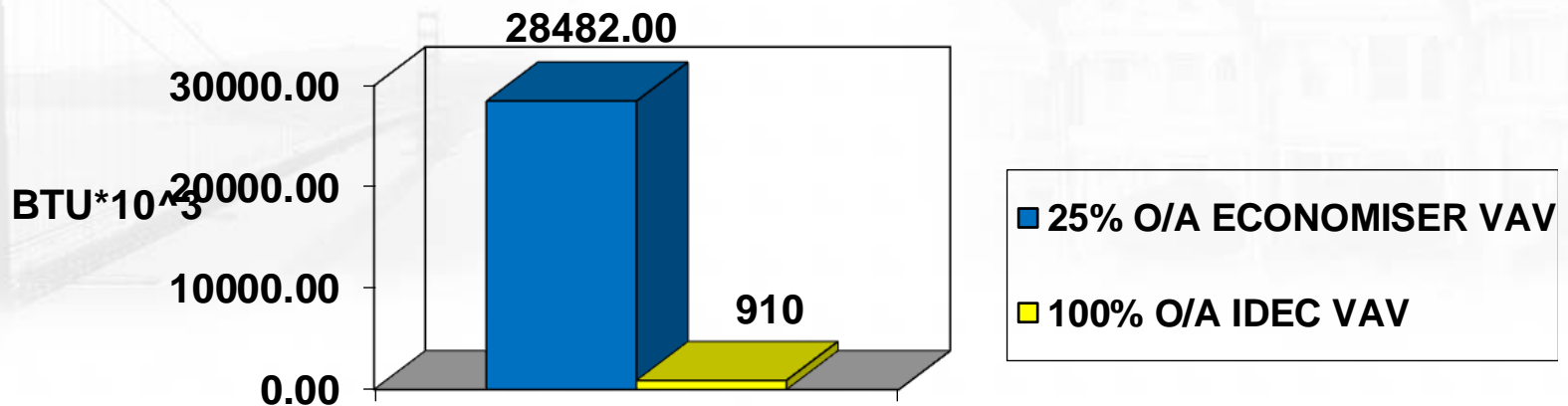
Peak kW at Dry bulb Design Condition of 100F/70F

For 10,000 cfm VAV



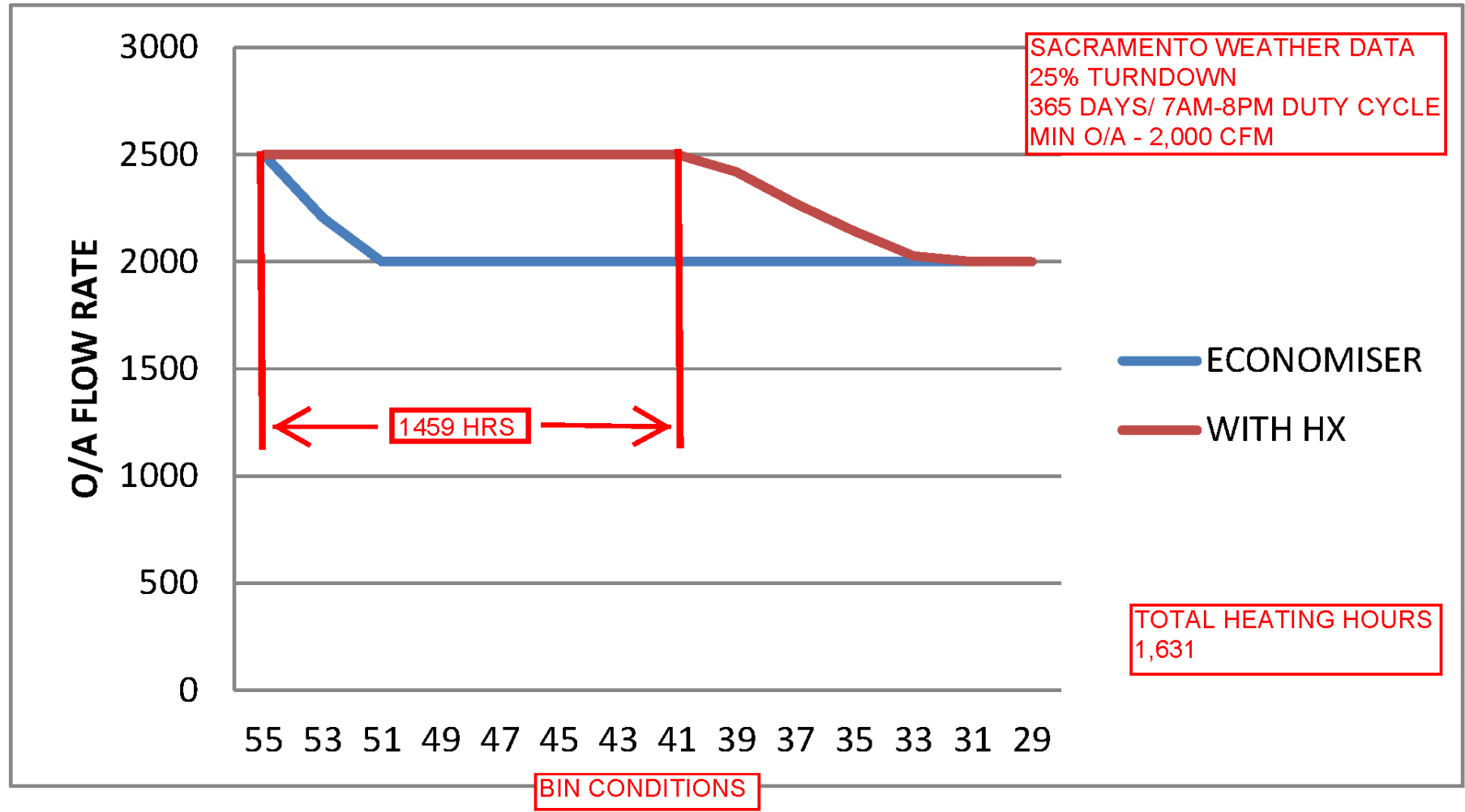
- Significant reduction in peak demand charges for the IDEC design.
- Further peak kW reductions possible by combining IDEC system with TES.

Annual Heating Requirement per 10,000 cfm Sacramento, CA@ 7am-8pm/365 Duty cycle

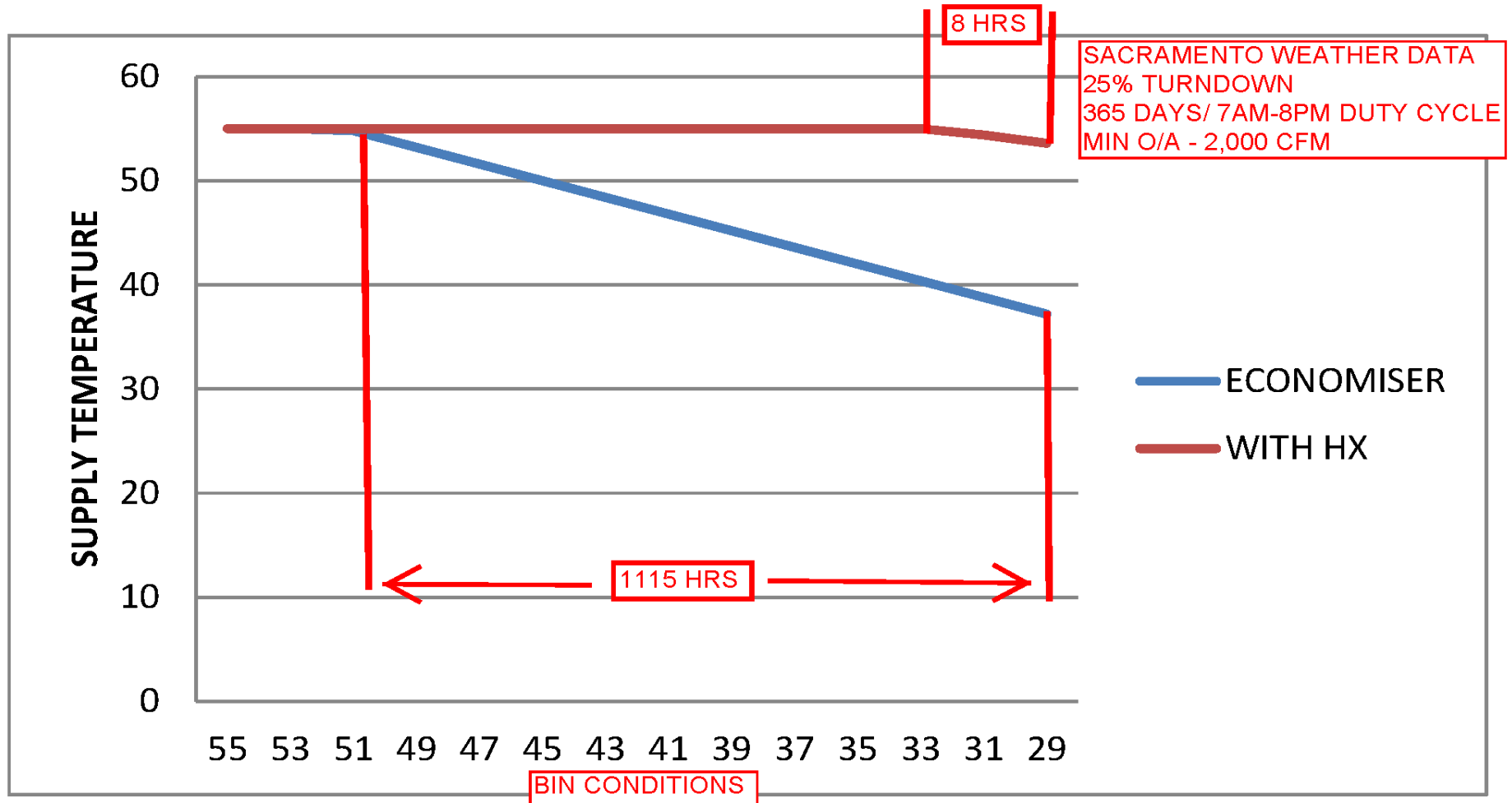


- VAV turndown considered for both systems.
- Significant reduction in heating requirements.

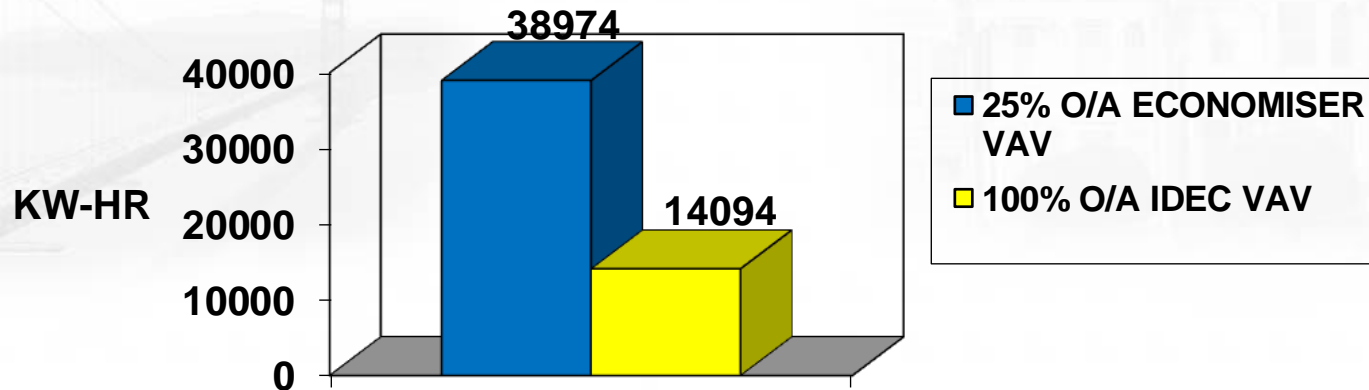
O/A FLOWRATE IN HEATING MODE



AUXILLARY HEATING REQUIREMENT



Annual KW-HR consumption per 10,000 cfm Sacramento, CA @ 7am-8pm/365 Duty cycle



- IDEC system KW HR consumption includes added fan KW to overcome pressure drops across Indirect/Direct components.

WATER CONSUMPTION

IEC VS CHILLER WITH COOLING TOWER

- Evaporation of 1 pound of water requires 1,000 BTU of heat
- To provide one ton of cooling, we need to remove 12,000btu/hr
- 1 ton of cooling requires 12lbs/hr of water consumption

Indirect Evaporative Cooler

Water Evaporation = 12 Lbs/hr =1.44 gallons/hr per ton of cooling

Cooling Tower

Chiller adds about 3,000 btu/hr of parasitic load for every 12,000 btu/hr of cooling (approx 25%)

Water Evaporation = 15 Lbs/hr =1.8 gallons/hr per ton of cooling

- Additional water is wasted in CT due to drift.
- Bleed rate is higher due to higher evaporation rate and low cycles of concentration

Water consumed at the power plant

- 2 gallons of water is required for every kWh of electrical power consumed

Chiller

- Efficiency of chiller is 0.8kW/ton
- 1 ton of cooling provided by chiller requires **1.6 gallons/hr** at the power plant

IEC

- Efficiency of IEC is 0.2kW/ton
- 1 ton of cooling provided by IEC requires **0.4 gallons/hr** at the power plant



business case

natural refrigerants

June 18-19, 2013 San Francisco

