

# **/// CO<sub>2</sub> as Refrigerant**

Challenges in use in railway air conditioning

/// 2017/09/26 - Lutz Boeck



- Motivation
- History activities of FT in the past
- Challenges by using CO<sub>2</sub> in passenger railways
  - Requirements for HVAC unit in railway application
  - Design and extreme conditions for different climatic zones
  - Performance & COP
  - Safety Concept
- Field trail
- Conclusion



# /// MOTIVATION FOR CO<sub>2</sub> AS REFRIGERANT



### /// MOTIVATION

- F-Gas-Regulation EU/517/2014 defines phase down
   → but no limitation of current used refrigerants for railway sector
- Influencing facts for moving forwards
  - **FT environmental philosophy**
  - Political decisions
  - **Customer inquiries**
  - **Progress in other industry**
- Enhancement of the internal know-how / preparation of the industrialization
- Project together with the NSB and SINTEF Modification of a FLIRT HVAC-unit from R134a to CO<sub>2</sub>
- → Ready for serial deliveries in 2020







## **/// HISTORY OF CO<sub>2</sub> IN FAIVELEY**

Retrospect

- **1995** First modification of existing compressors for  $CO_2$ -Application, implementation in a unit for lab test
- **2000** Development of test rig for  $CO_2$  lab test activities
- 2005 Development and construction of a drivers cab unit with CO<sub>2</sub> → Tram Hannover ÜSTRA
- 2005-2011 Successful operation
- **2014** Restart of activities Study of different circuit layouts, potential suppliers, control strategies and safety concept
- **2015** Start of research project with SINTEF and NSB

First tram unit 1995

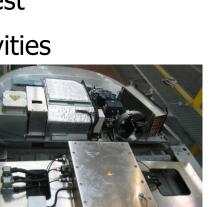






Component test rig 2005





CO<sub>2</sub> unit on ÜSTRA tram 2005



### /// HISTORY OF CO<sub>2</sub> IN FAIVELEY

Lessons learned



### Lessons learned from projects - Cycle improvements

high pressure control
accumulator
internal heat exchanger
"Safe" evaporators
minimization of refrigerant mass
Heat pump function

All these features are now integrated in NSB demonstrator!





Roof installation on ÜSTRA tram 2005



# /// CHALLENGES





Railway specific requirements

### CO<sub>2</sub> unit shall fulfil similiar parameters as for R134a units as

- Cooling capacity (at design and extreme conditions)
- Heating capacity (down to -20° C)

### Dimensions

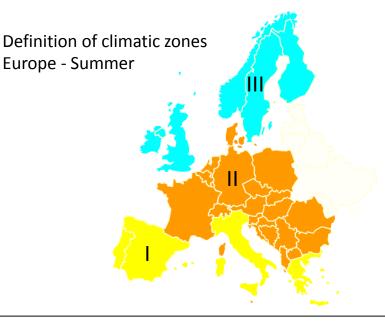
(limited space for unit and components  $\rightarrow$  large  $\Delta t$  on HX)

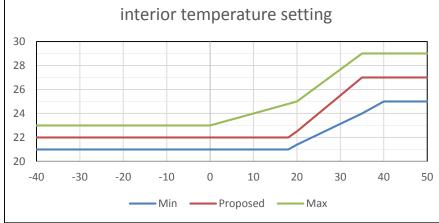
- > Weight
- Shock & vibration proof (EN 61373:2011)
- Energy consumption (Peak, Efficiency, Yearly energy consumption)
- Reliability
- Maintenance
- Safety
- Stillstand pressure (train parked without power)
- 9 /// ATMOsphere Europe / Berlin / 25-27 September 2017





Design and extreme conditions for different climatic zones - EN 13129





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#### Design conditions

	Winter	Summer		
Zone	Minimum exterior temperatures °C	Maximum exterior temperatures °C	Relative humidity [%]	Equivalent solar load (En) W/m²
I	-10	+ 40	40	800
п	-20	+ 35	50	700
ш	-40	+ 28	45	600

#### **Extreme conditions**

	Winter	Summer			
Zone	Minimum exterior temperatures	Maximum exterior temperatures	Relative humidity	Equivalent solar load (En)	
	°C	°C	%	W/m <sup>2</sup>	
I	-15	+ 45	30	800	
п	-25	+ 40	40	700	
ш	-45	+ 33	30	600	

#### **Operational limiting conditions**

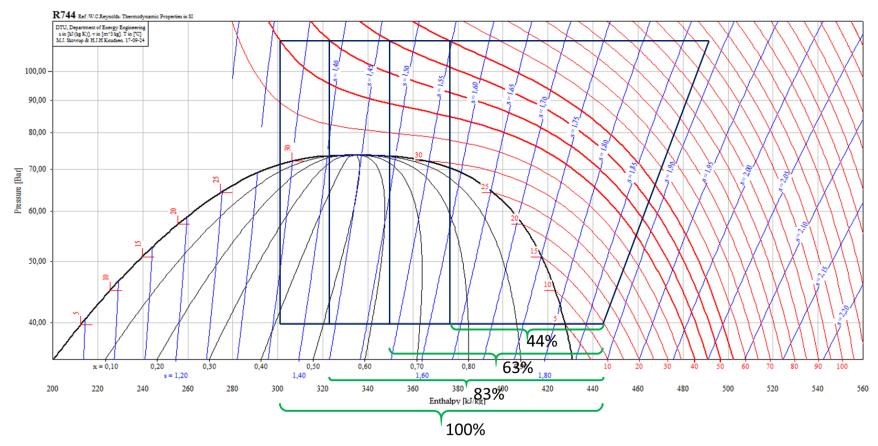
	Summer				
Zone	Maximum exterior temperatures	Relative humidity	Equivalent solar load (En)		
	°C	96	W/m <sup>2</sup>		
I	+ 50	25	800		
п	+ 45	30	700		
ш	+ 38	25	600		





Performance and COP – depending from ambient temperature

Reduction of cooling capacity depending from gascooler outlet temperature  $40/45/50/55^{\circ}$  C at given max. pressure



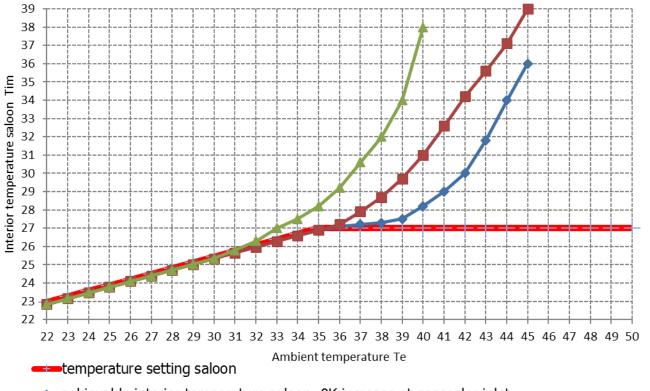




Design Conditions e.g. climatic zone II – EN 13129 / EN 14750

#### Interior saloon temperature in dependency from ambient temperature / gascooler inlet temperature

(te=35/40/45°C)



<sup>-</sup> very sensitive - Significant

to be considered in design phase

-----achievable interior temperature saloon, 0K increase at gascooler inlet

- achievable interior temperature saloon, 3K increase at gascooler inlet
- -----achievable interior temperature saloon, 6K increase at gascooler inlet





Safety

CO<sub>2</sub> unit typically classified in Category II (I) PED 2014/68/EG

### Safety concept at different stages on product life cycle required:

Manufacturer Car builder Operator

- Manufacturer  $\rightarrow$  safety concept for manufacturing and serial test
  - $\rightarrow$  safety concept for installation on train
  - → safety concept for operation, maintenance and de-commissioning
- $\rightarrow$  Protection against hazards associated with:
  - Physical and chemical characteristics of refrigerants
    - Refrigerant charge limitation
    - Machinery room requirements
  - Pressures and temperatures occurring in the refrigerant cycle
    - Requirements for components and piping
    - Requirements for assemblies

VDB Guideline - R744-Klimaanlagen in Schienenfahrzeugen

Technische Auslegungsgrundlagen und Sicherheitsnachweise



# /// FIELD TRAIL



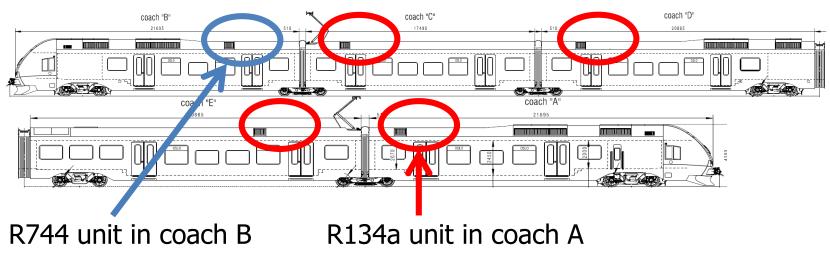
**ATMO** sphere

 $CO_2$  unit for train – in service trial

### Operation of a R744 unit under real service conditions

- Train of type "FLIRT" at NSB network around Oslo in Norway
- Test period 2017/18  $\rightarrow$  12 Months
  - ightarrow Evaluation of a complete cooling and heating cycle
- Data recording for R744 / R134a units on the train
  - operation of both systems under identical conditions
  - $\rightarrow$  direct comparison possible







 $CO_2$  unit for train – in service trial

- Identical cooling / heating capacity in design points
- Similiar fresh air flow at the different operating conditions
- Identical dimensions, mechanical, electrical and air interfaces
- No modification of maximum electrical input power
- Increase of weight by max. 20% acceptable (Axle load)

- 2 circuit system (Q<sub>0</sub>=18 kW at design point 28° C/60%r.H)
- Changeover between cooling and heat pump operation







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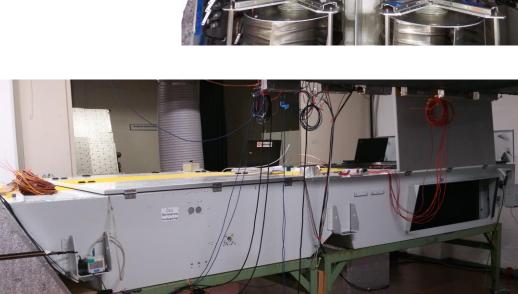


 $CO_2$  unit for train – in service trial

### Lab tests:

- Pressure test HP=132 bar / LP=111 bar
- Shock and vibration
- Functional test safety devices
- Design point Heating / Cooling
- Extreme conditions Cooling
- Maximum conditions Cooling
- Part load conditions
- Changing conditions
- Changeover AC  $\leftrightarrow$  HP







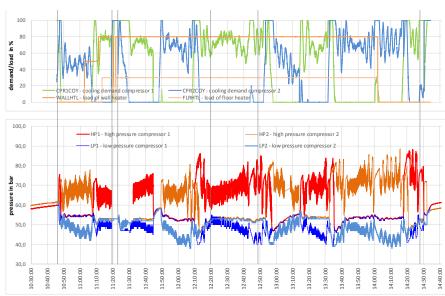




#### Field tests impressions







ATTENTION - CO<sub>2</sub>Test unit Service of CO<sub>2</sub> refrigeration circuit by trained staff only!

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# /// CONCLUSION





Conlusion

## CO<sub>2</sub> versus R134a unit

- Stronger reduction in cooling capacity at conditions above design point, more sensitive on performance degradation of HX
- More complex circuit layout  $\rightarrow$  higher number of components
- Practical useable operating pressure limited due to stepwise arrangement of safety elements (3 steps)
- Management of standstill pressure passive through Accumulator
- Weight level appr. 20% higher than R134a system (Accu, Comp.)
- Cost level significant higher
- Certification more expensive  $\rightarrow$  Category II acc. PED 2014/68/EU
- Safety concept, especially for small rooms (DC, toilet)
- Control strategy different
- Power supply adaption required
- Yearly energy consumption ??





Future / Potential for improvement / Tasks

- ✓ Solution for warm / extreme warm climatic zones
- $\checkmark$  Adaption of design points (to cover stronger reduction at high temp.)
- ✓ System design for heating and cooling (to compensate lower COP in cooling at high ambient conditions in yearly energy consumption)
- ✓ Design of HX for smaller temperature difference (dimensions, weight, air flow ↗)
- ✓ Increase of practical useable operating pressure
- ✓ Use of ejectors to compensate degradation in efficiency at high ambient conditions
- Management of stillstand pressure at hot and sunny days (no parking mode without power)
- $\checkmark\,$  Amplification of train power supply for climatic zone I
- ✓ Training of staff for maintenance and repair



Thank you for your attention.

